



# Lower limit of weak flame in a heated channel

Yosuke Tsuboi, Takeshi Yokomori\*, Kaoru Maruta

IFS, Tohoku University

\*Keio University

For highly efficient combustion, HiCOT, HCCI, etc...

- Controlling initial state of combustion important
- Exergy (Availability) efficiencies of combustion process significantly improved

## **General characteristics of combustion with heat recirculation**

studied in the context of microcombustion (Maruta et al. PCI 30, 2005)

- 1-D straight meso-scale channel with temperature gradient
  - **S-shaped curve** (Normal and **weak** flames identified)
  - *Flames with repetitive extinction and ignition* (FREI) identified

# Background

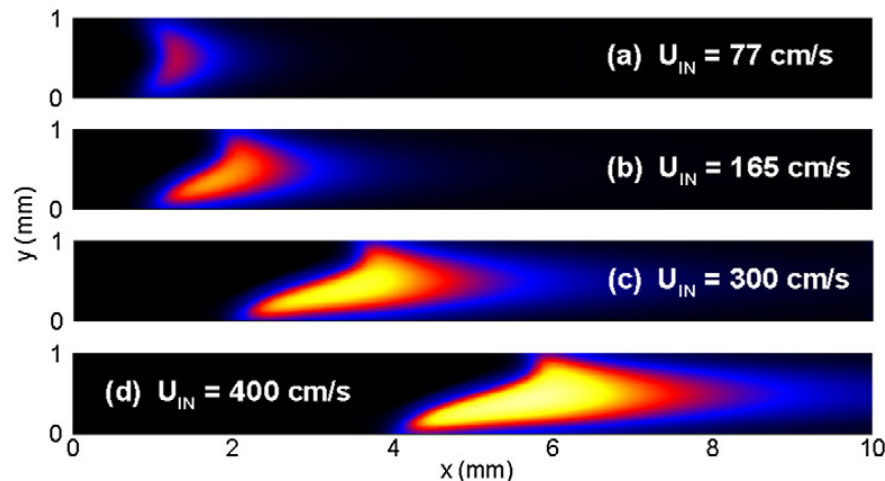
- 1-D straight meso-scale channel with temperature gradient covered by:

-Dynamics of flame in narrow tube examined

(Jackson, Buckmaster, Lu, Kyritsis, Massa, PCI31, 2007)

- DNS study, Hydrogen/air flames confirmed details of FREI  
Flames in high velocity region highlighted

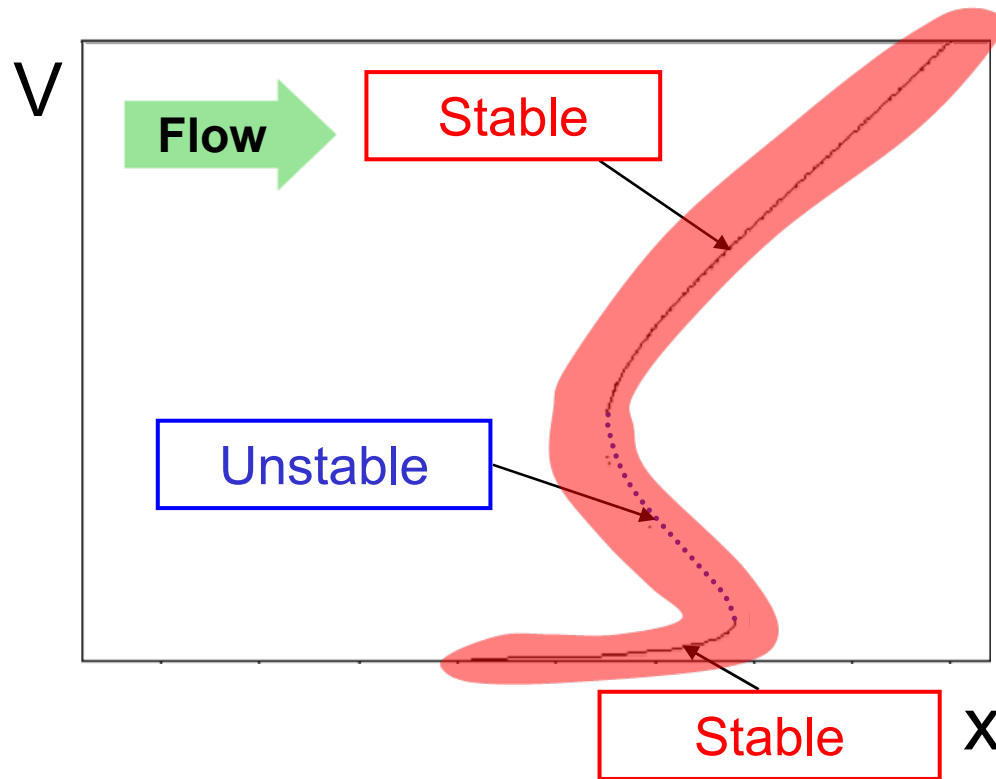
(Pizza, Frouzakis, Mantzaras, Tomboulides, Broulouchos CNF152, 2008)



- Further understandings of weak flame characteristics required
  - Weak flame limit ?
  - Relation with ignition and weak flame ?

# Overviews of our completed studies

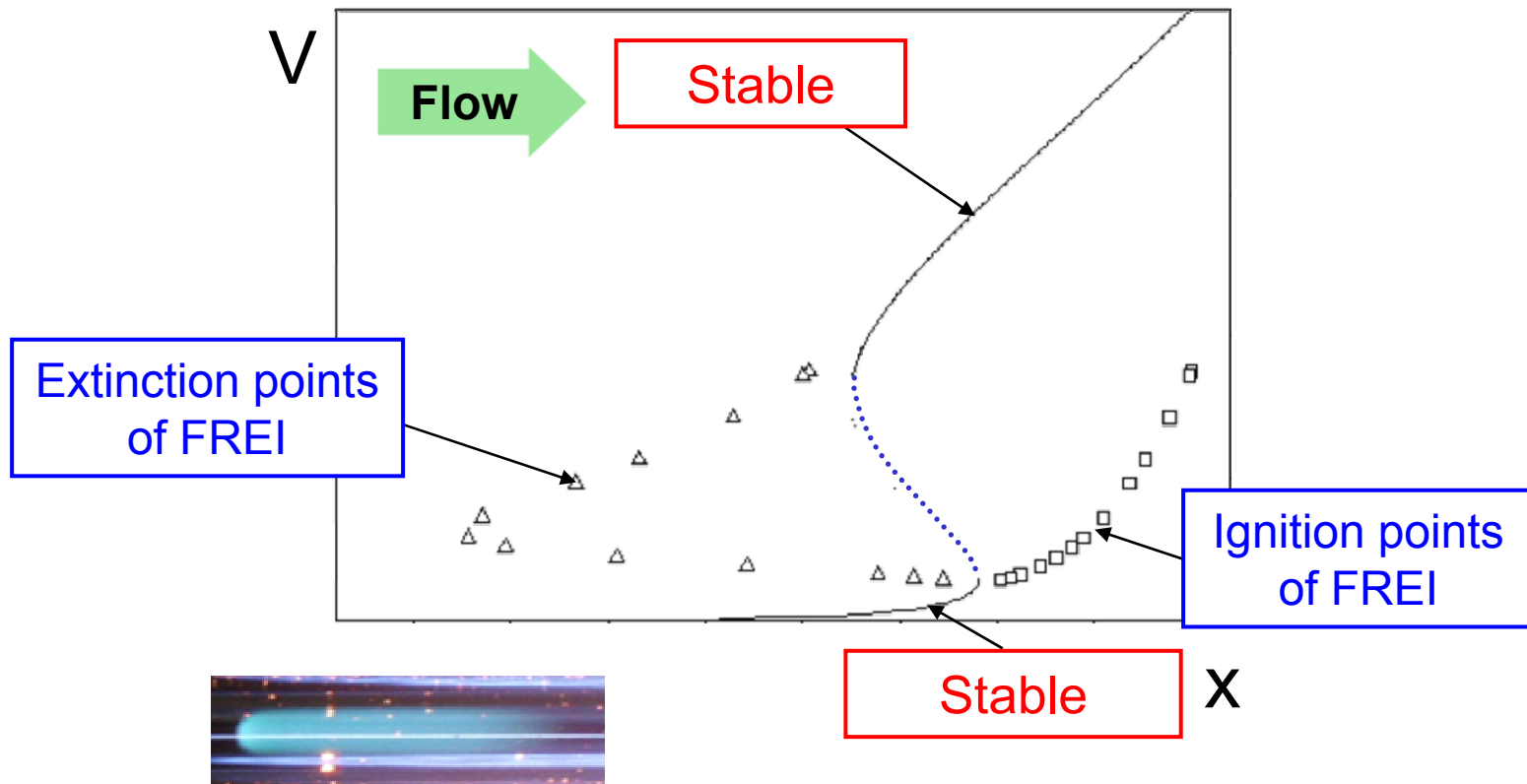
Overall picture of flame responses understood by linear stability analysis



**S-shaped** flame responses

(Maruta et al. CESW40, 2004: PCI 30, 2005)

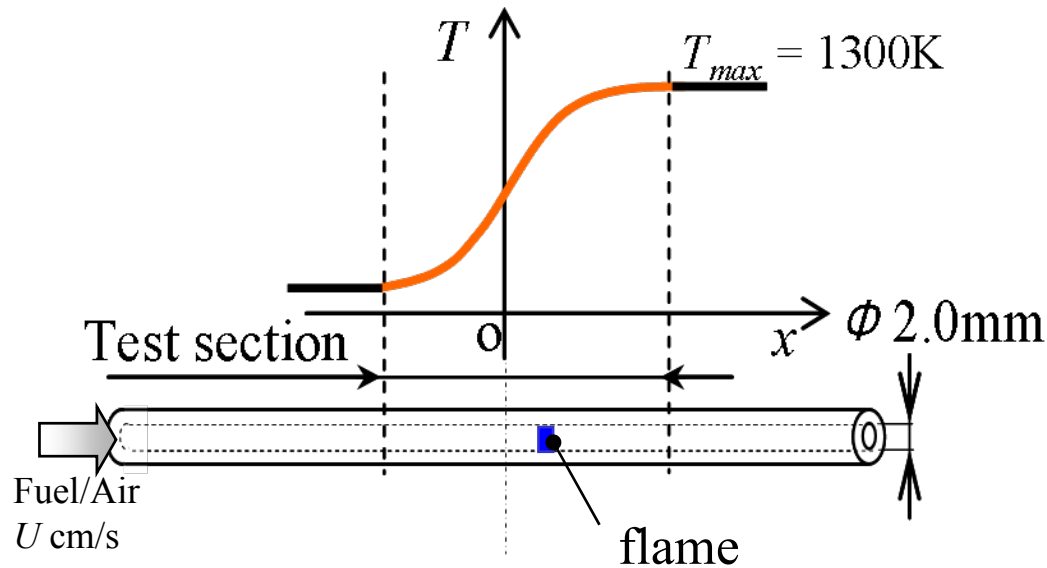
# Overviews of our completed studies



Unstable middle branch confirmed to correspond flames with repetitive ignition and extinction (FREI) (Minaev et al., CTM 2007)

# Experimental apparatus

Wall temperature profile established by an external heat source



Meso-scale channel

Quartz glass tube  
 $d=2.0\text{mm}$ ,  $L=300\text{mm}$

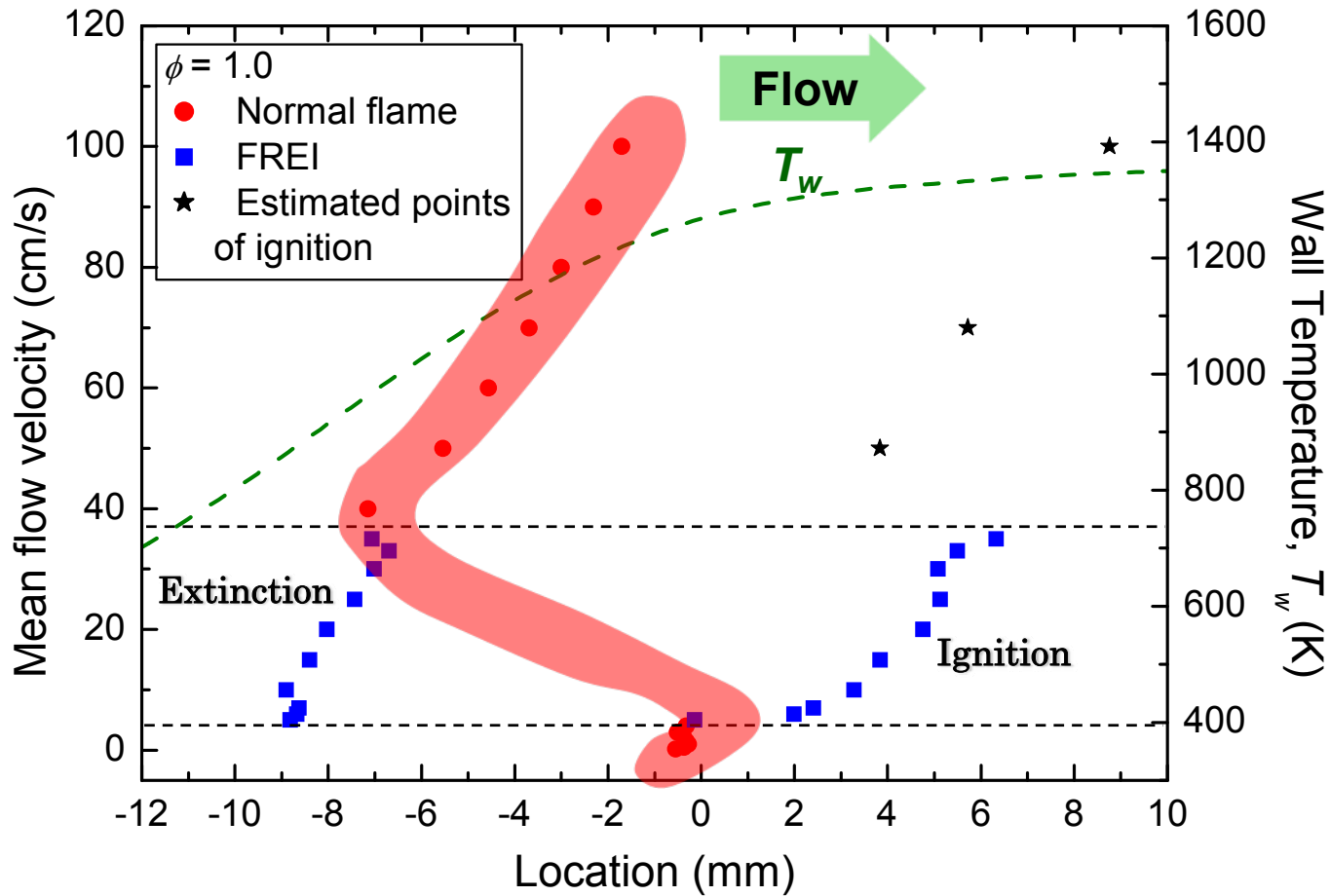
Mixture

Channel : Methane / Air

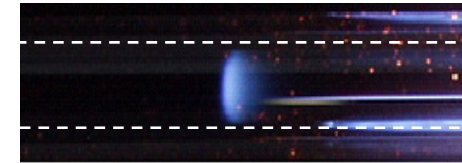
Visualization

Digital still camera  
with CH filter

# Experimental results ~ Flame responses ~

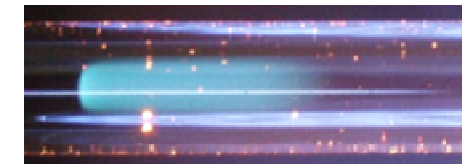


(1) Normal



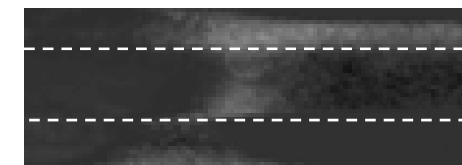
V=50cm/s

(2) FREI



V=20cm/s

(3) Normal



V=0.2cm/s

Wall Temperature,  $T_w$  (K)

High and low flow velocity regions



Normal flame

Middle velocity region



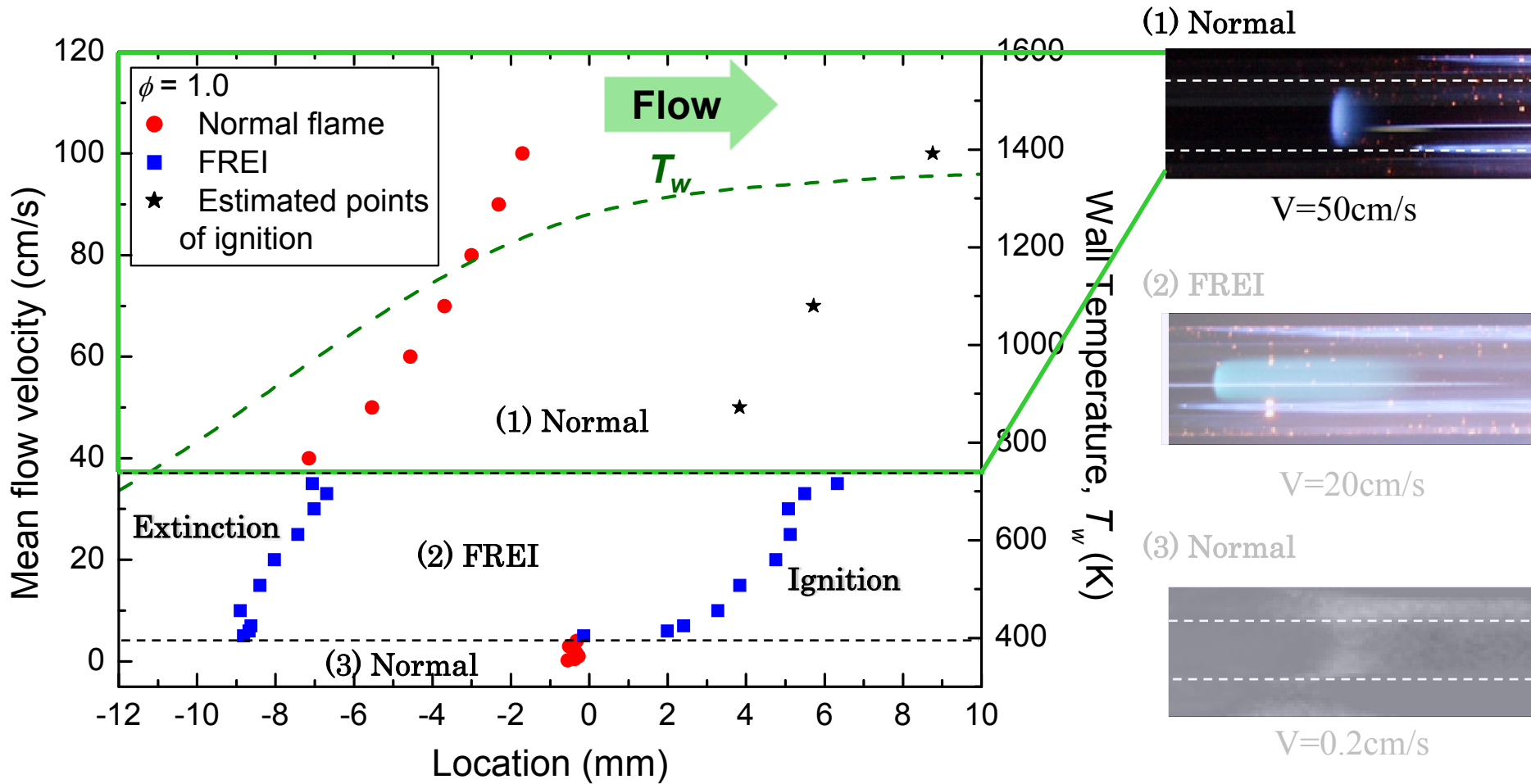
FREI



**S-shaped flame responses**

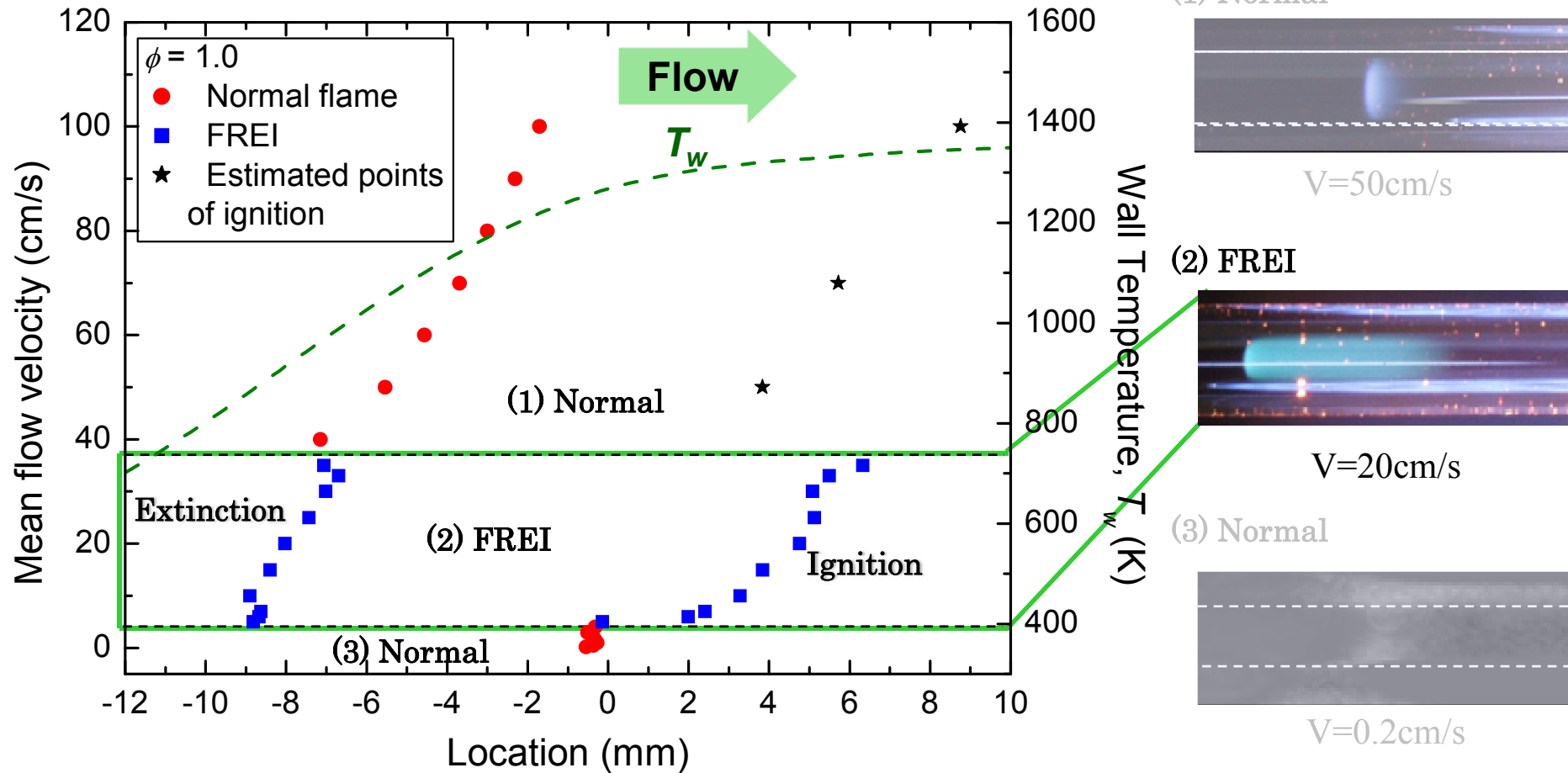


# Experiments ~ Higher velocity region ~



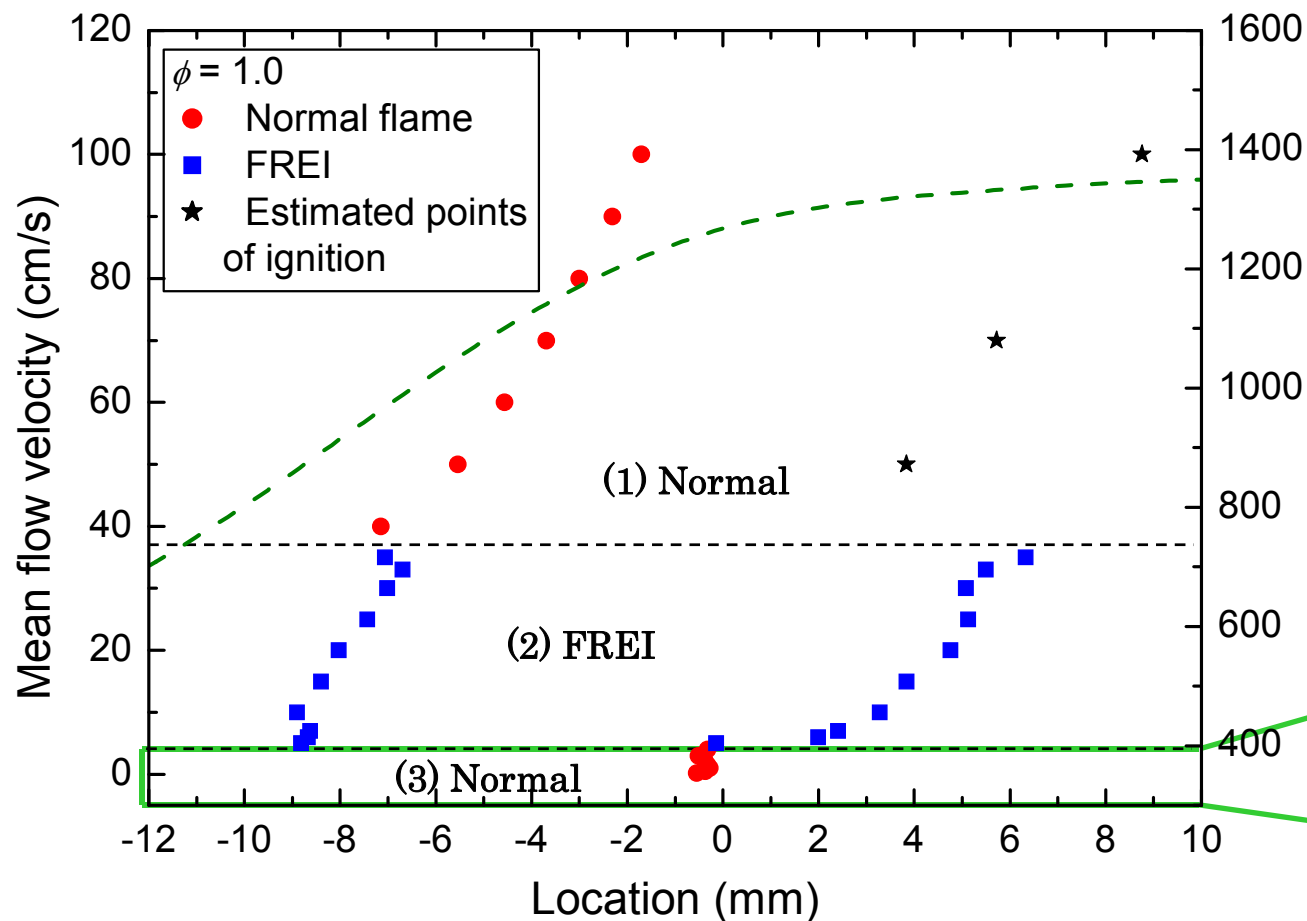
**Self-ignition at downstream side  $\rightarrow$  stabilized at upstream location**  
**Flame position shifts toward upstream with the decrease of flow velocity**

# Experiments ~ Moderate velocity region ~

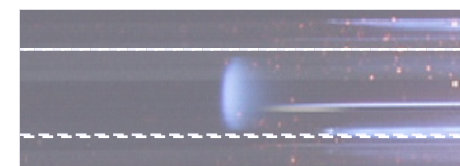


**Ignition in downstream → Flame kernel moves toward the upstream**  
**← Flame extinction due to low temperature of the wall ←**

# Experiments ~ Lower velocity region ~



(1) Normal



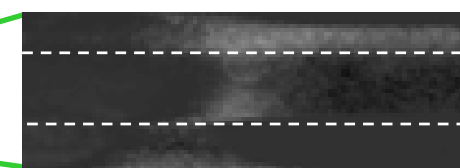
$V=50\text{cm/s}$

(2) FREI



$V=20\text{cm/s}$

(3) Normal



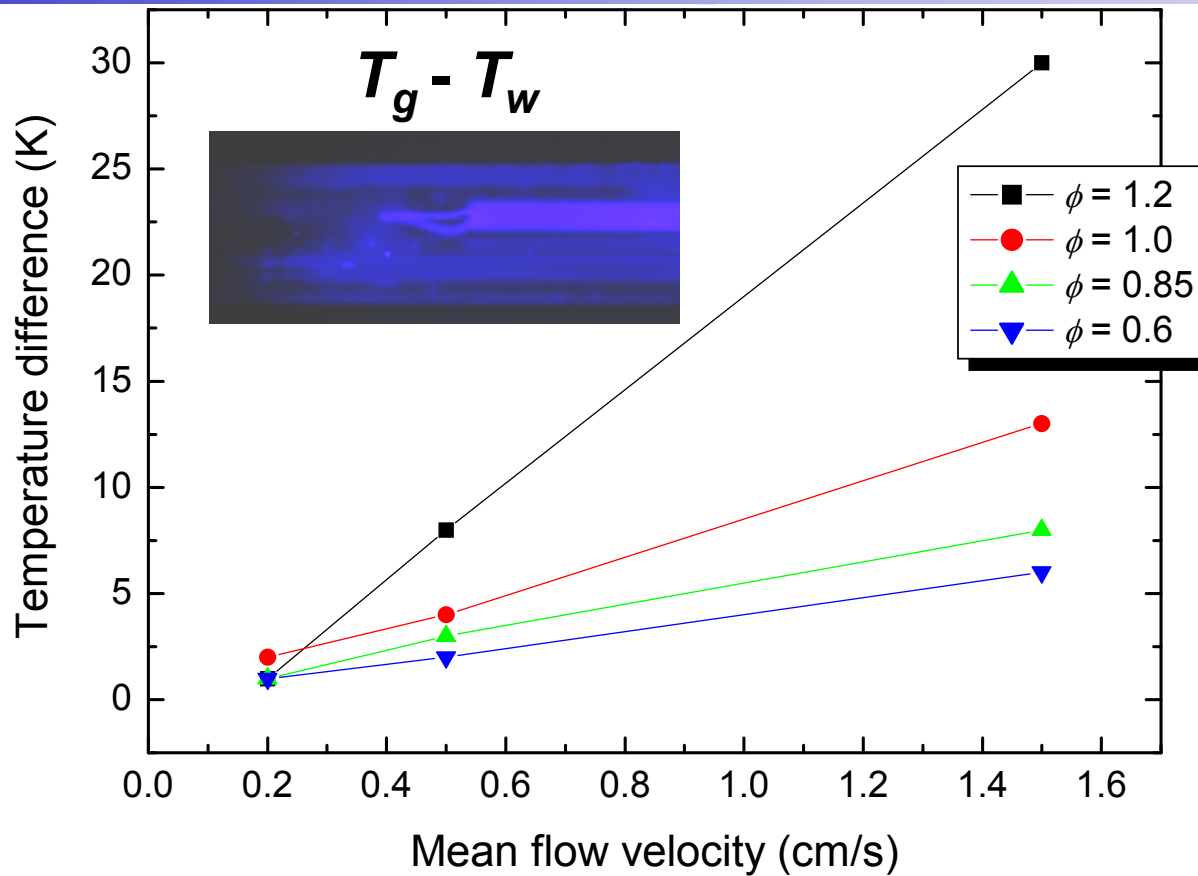
$V=0.2\text{cm/s}$

(Exposure time : 1200 s)

$V_m < 0.2 \text{ cm/s}$  → No luminous flame → **Lowest burning velocity?**

Flame is stabilized near the ignition position with very small heat release

# Experiments ~ Gas and wall temperature differences ~



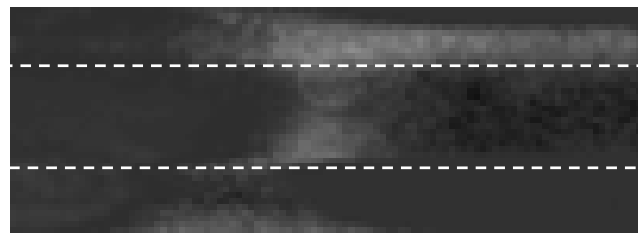
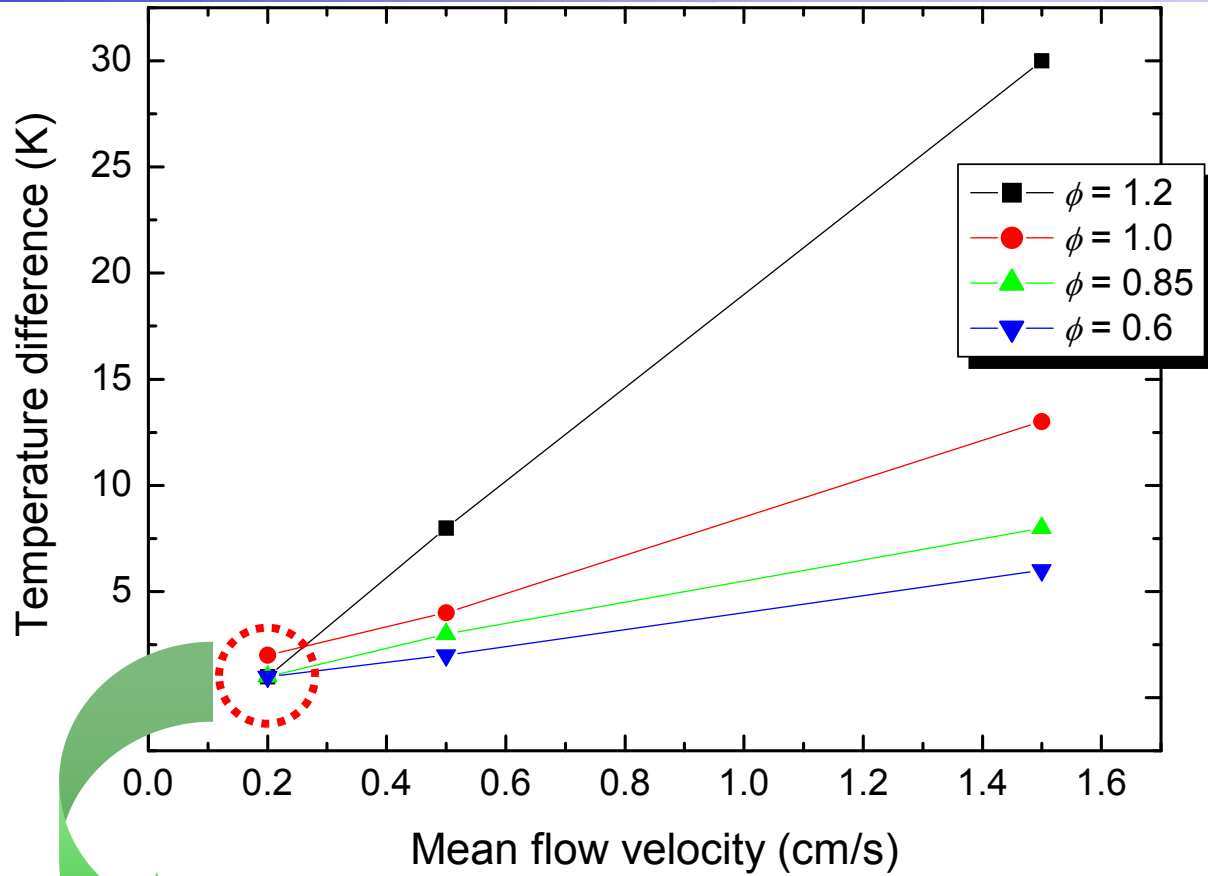
$V = 0.2$  cm/sec,  $T_w = 1225$  K  $\Rightarrow$   $T_g - T_w < 2$  K

➔ Small temperature increase + Flame position of weak flame close to an ignition point

Lowest flame temperature corresponds to ignition temperature

Universal ignition temperature can be identified

# Experiments ~ Gas and wall temperature differences ~



**Mechanism of lower limit of weak flame?**

# Computations

Calculation code

**1-D steady state flame code** ( Kee et al., 1989 )

Energy equation

$$\dot{M} \frac{dT}{dx} - \frac{1}{c_p} \frac{d}{dx} \left( \lambda A \frac{dT}{dx} \right) + \frac{A}{c_p} \sum_{k=1}^K \rho Y_k V_k c_{pk} \frac{dT}{dx} + \frac{A}{c_p} \sum_{k=1}^K \dot{\omega}_k h_k W_k - \frac{A}{c_p} \frac{4\lambda Nu}{d^2} (T_w - T) = 0$$

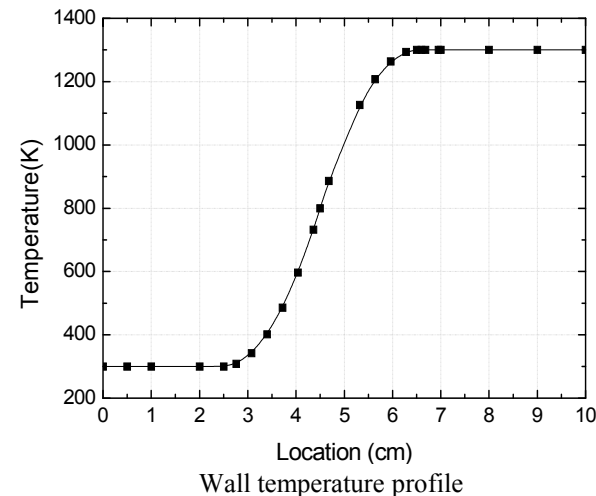
Convective heat transfer

Chemistry

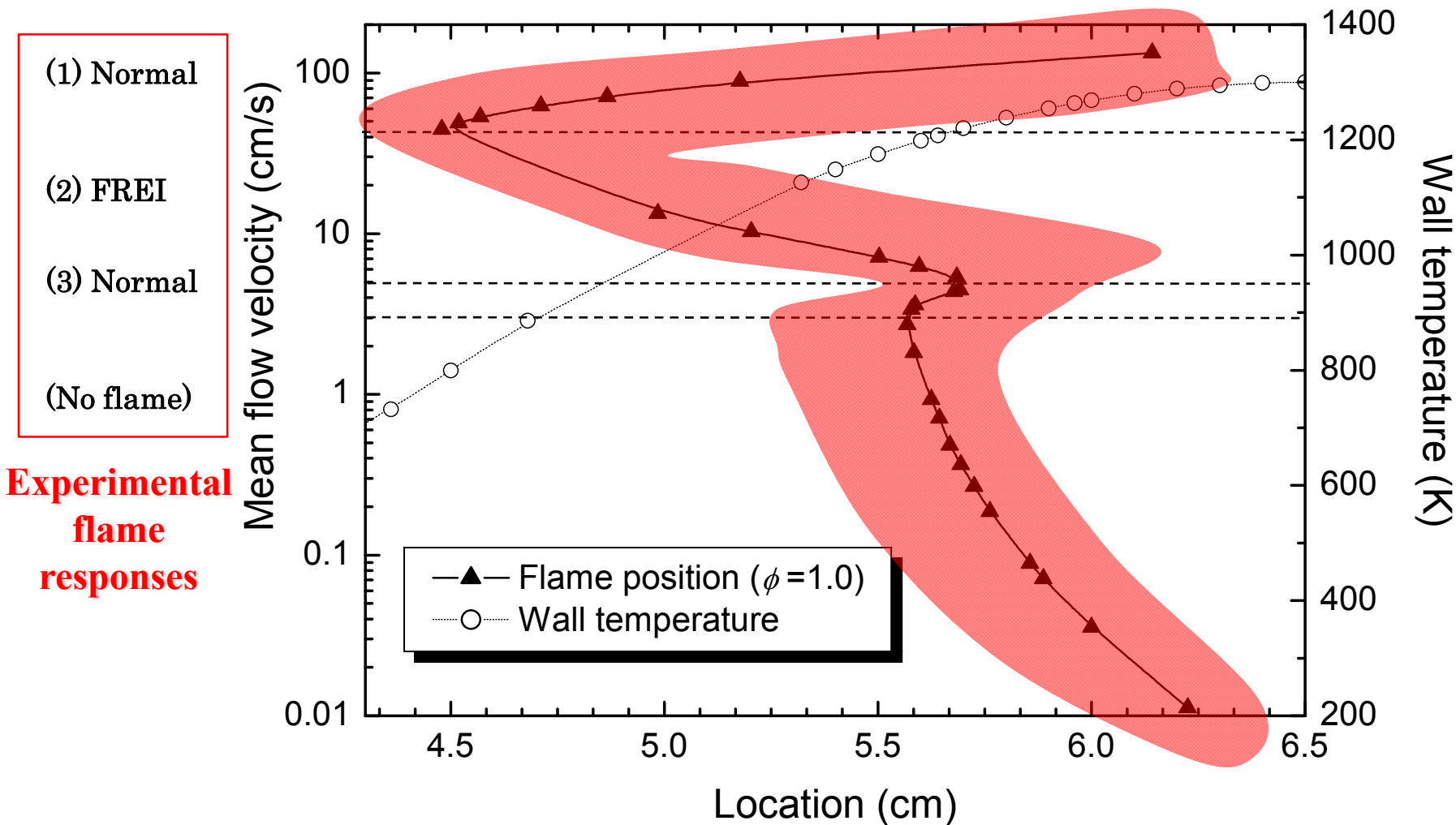
**GRI-mech 3.0**

Assumption

- Fixed wall temperature
- $Nu = 4$  (Constant)



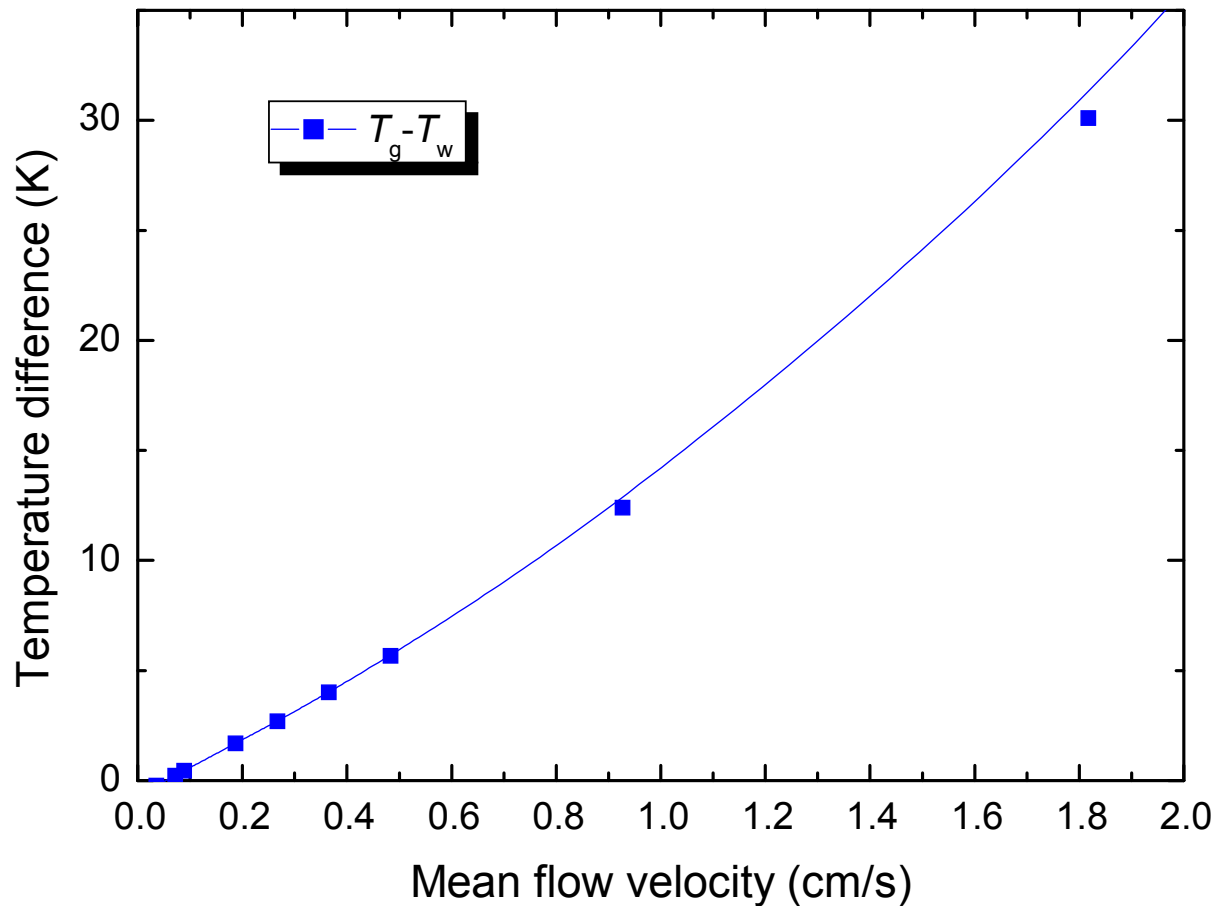
# Computations ~ Flame responses ~



**Two stable and two unstable solutions agreed with experimental regimes**

**Flame responses exhibit  $\epsilon$ -shape  $\rightarrow$  Limit in lower stable branch**

# Computations ~ Gas and wall temperature difference ~



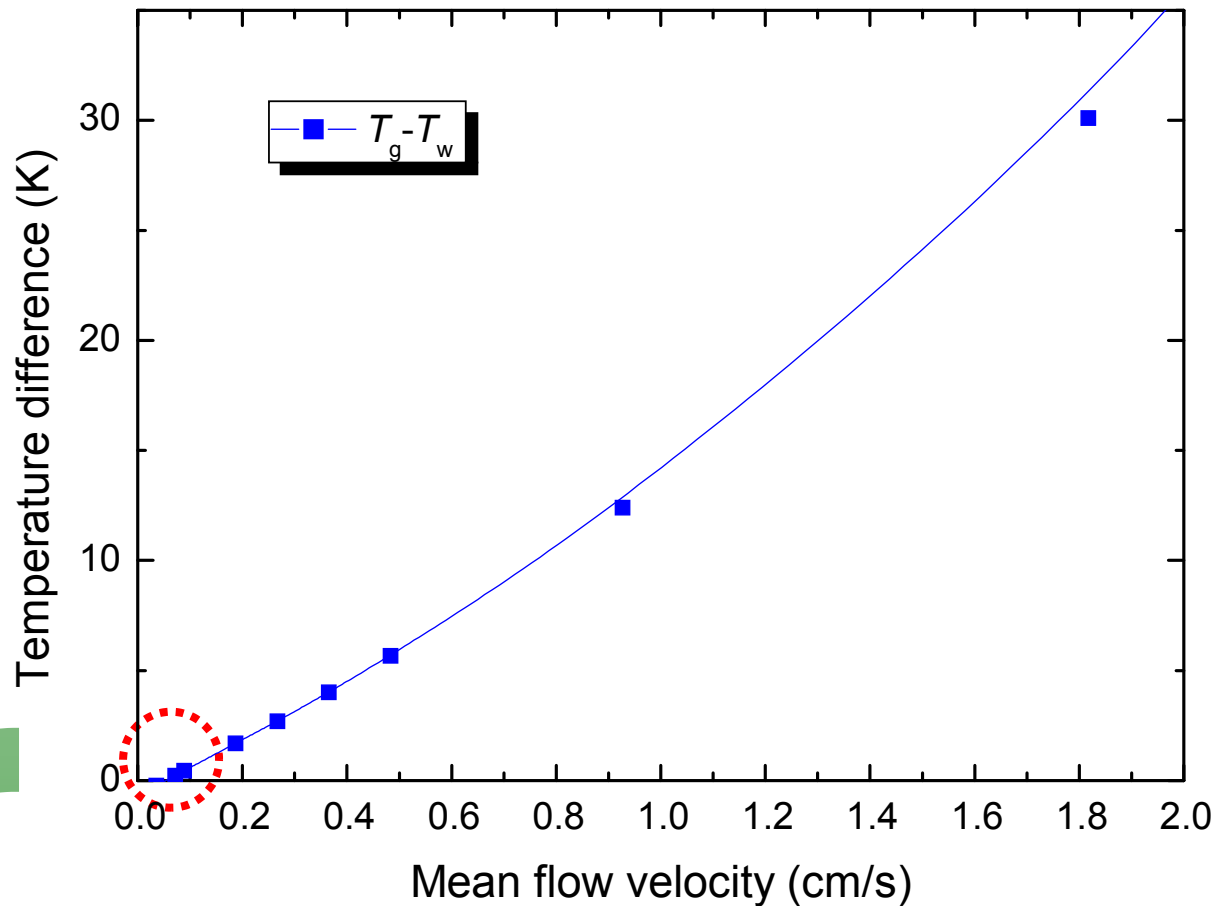
**Difference becomes smaller with the decrease of flow velocity**

$T_g - T_w < 2 \text{ K} \Rightarrow V = 0.1 \text{ cm/s}, T_w = 1230 \text{ K}$

**There exists the lowest burning velocity**



# Computations ~ Gas and wall temperature difference ~



## Mechanism of lower limit of weak flame?

Our assumption: limit induced by mass dissipation

## Mass transport of light radical (OH)

Convective mass flux



$$\rho Y_{OH} V_m$$

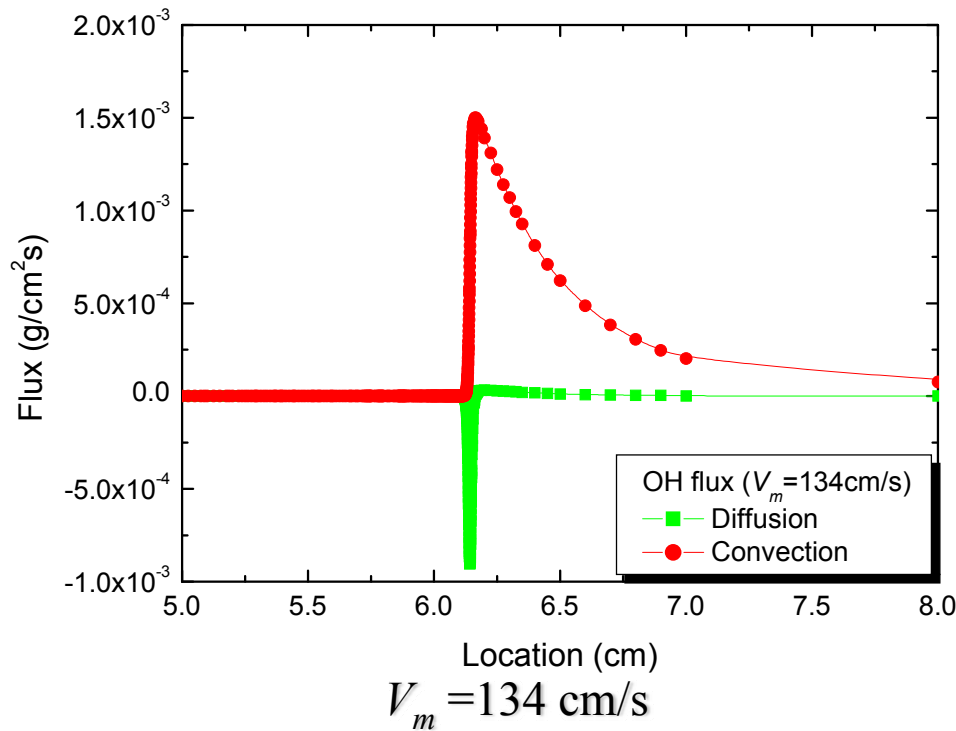
$V_m$ : Mean flow velocity

Diffusive mass flux

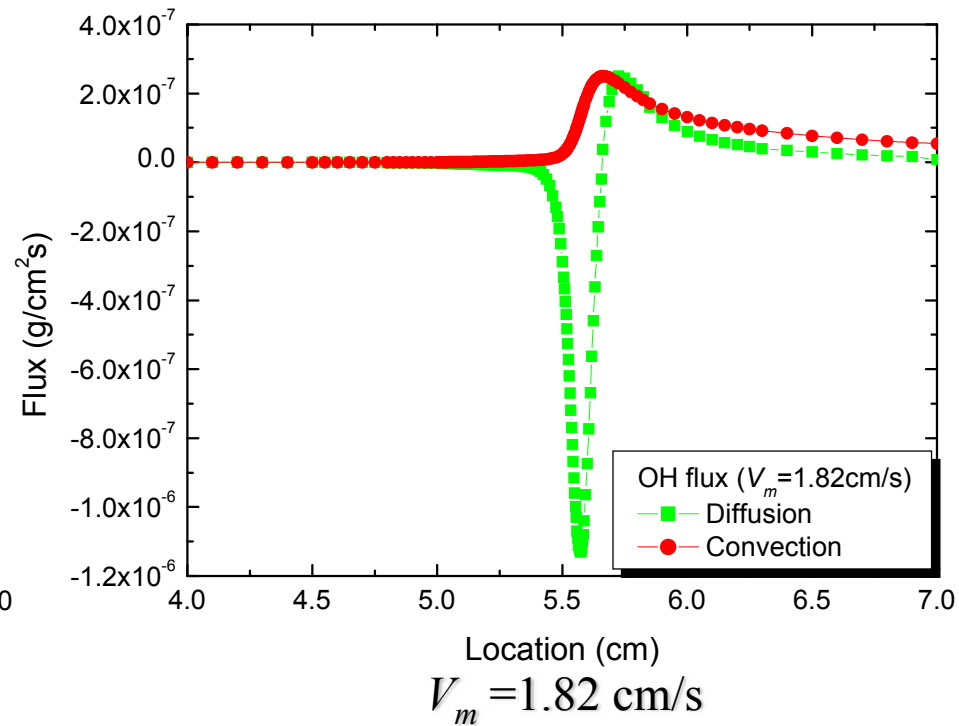


$$\rho Y_{OH} V_{OH}$$

$V_{OH}$ : Diffusion velocity

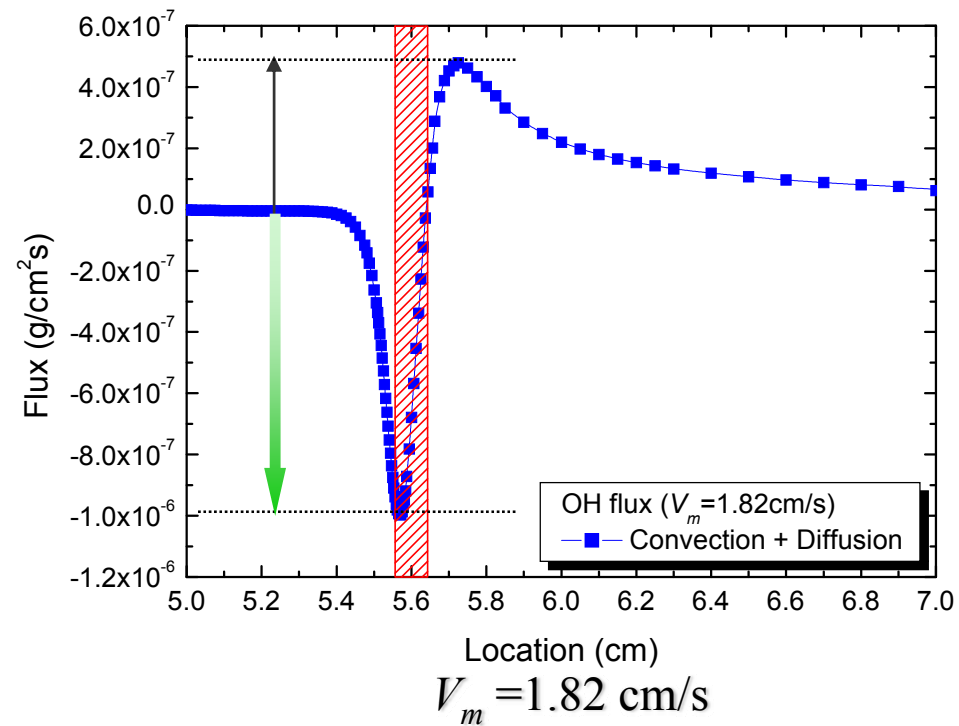
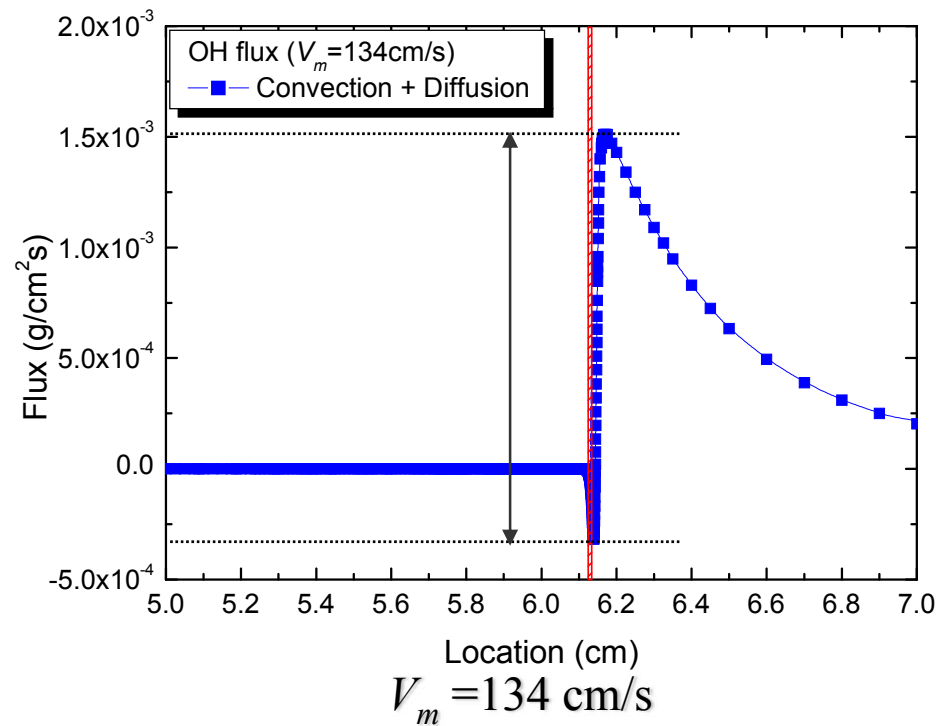


Convection > Diffusion



Convection < Diffusion

## Total mass fluxes



Mass dissipation in low velocity region significantly larger than that in higher velocity region: radicals diffuse out

**Lower limit of weak flame induced by diffusion effects**

# Conclusions

- Existence of **lower limit of weak flame** confirmed experimentally even if heat loss compensated by an external heating.
- Lowest flame temperature corresponded to **ignition temperature**.
- Computed flame responses exhibited  **$\epsilon$ -shape**, which implies existence of lower limit of weak flame in a lower stable branch.
- Based on computations, mechanism of lower limit of weak flame is supposed to be **mass dissipation loss** due to **mass diffusion** in low velocity region.